Lecture 10. THE CALCULATION OF THE BAR ON STRENGTH AND RIGIDITY BY TORSION

Assos. Prof. A. Kutsenko

Plan of lecture

- 1. The determination of the inside and outside diameter of the shaft from on rigidity by torsion
- 2. Typical example of calculation of shaft by torsion
- 3. The statically indeterminate problem under torsion

Given shaft has:

- L=3 m T=25 kNm
- G = 3 GPa $[\tau] = 90 MPa$

Determine the inside and outside diameter of the shaft

The angle of twist is equal: $\theta = \frac{T\ell}{GI_{\rho}}$ Thus, in the 3-m length we have:

$$\frac{2,5}{57,3} = \frac{25000 \cdot 3}{85 \cdot 10^9 \frac{\pi}{32} \left(D^4 - d^4 \right)}$$

or:

$$D^4 - d^4 = 206 \cdot 10^{-6} \text{ m}^3$$

From Eq. (9.2), we get: 90 $\cdot 10^6 = \frac{25000 \cdot D/2}{\frac{\pi}{32} \left(D^4 - d^4 \right)}$ or:

$$D^4 - d^4 = 1414 \cdot D \cdot 10^{-6}$$

Comparison of the right-hand sides of these equations indicates that:

$$206 \cdot 10^{-6} = 1414 \cdot D \cdot 10^{-6}$$

Thus, finally we get that:

or:
$$D = 0,145 \text{ m}$$

 $D = 145 \text{ mm}$

Substitution of this value into either of the equations then gives:

$$d = 0,125$$
 m

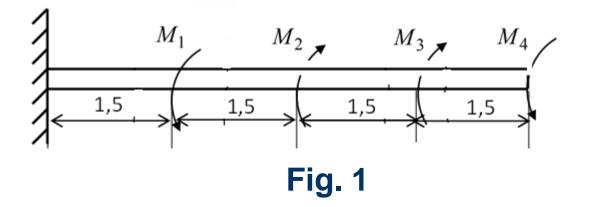
or:

d = 125 mm

Given shaft has:

$$M_1 = M_2 = M_3 = M_4 = 150$$
Nm

 $[\tau] = 55 \text{ kPa}$



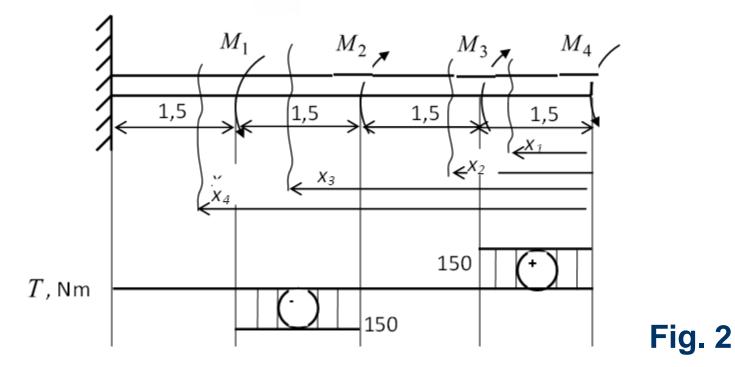
Drawing the diagram of torques T I portion $0 \le x_1 \le 1.5$ m $T_1 = M_4 = 150$ Nm

Il portion $1,5 \le x_2 \le 3$ m $T_2 = M_4 - M_3 = 0$ Nm

III portion $3 \le x_3 \le 4,5 \text{ m}$ $T_3 = M_4 - M_3 - M_2 = -150 \text{ Nm}$ IV portion $4,5 \le x_4 \le 6 \text{ m}$

$$T_4 = M_4 - M_3 - M_2 + M_1 = 0$$

Diagram of torques T



From the condition of durability the diameter of shaft is:

$$d \ge 3 \sqrt{\frac{16 \cdot T_{\max}}{\pi \cdot [\tau]}}$$

in this case we have: $T_{\text{max}} = 150$ Nm

so
$$d \ge 3 \sqrt{\frac{16 \cdot 150}{3,14 \cdot 55 \cdot 10^3}} \approx 0,024 \text{ m}$$

Thus, d = 30 mm

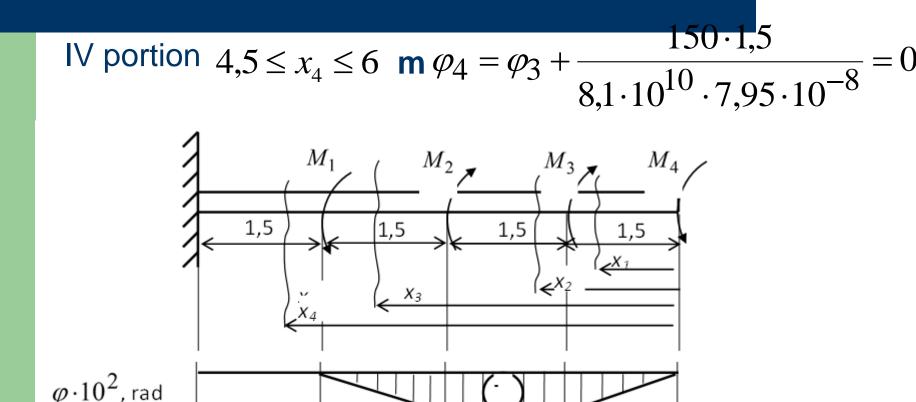
and
$$I_{\rho} = \frac{\pi d^4}{32} = \frac{3,14 \cdot 3^4}{32} \approx 7,95 \text{ cm}$$

Drawing the diagram of the angle of twist I portion $0 \le x_1 \le 1.5$ m $\varphi_1 = 0$

Il portion
$$1.5 \le x_2 \le 3$$
 m
 $\varphi_2 = \frac{-150 \cdot 1.5}{8.1 \cdot 10^{10} \cdot 7.95 \cdot 10^{-8}} \approx 3.49 \cdot 10^{-2}$ rad

III portion $3 \le x_3 \le 4,5$ m

$$\varphi_3 = \varphi_2 = 3,49 \cdot 10^{-2}$$
 rad



3,49

Fig. 3

Let us determine the reactive torques at the fixed ends of the circular shaft loaded by the couples

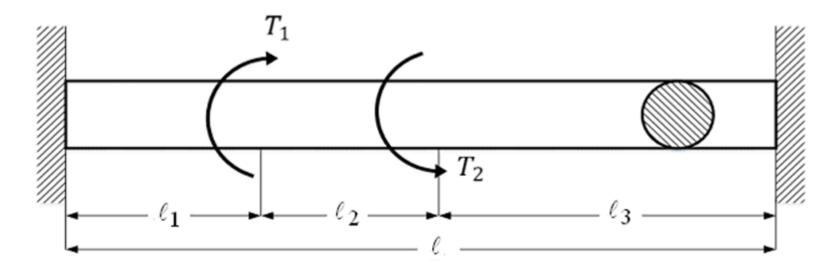
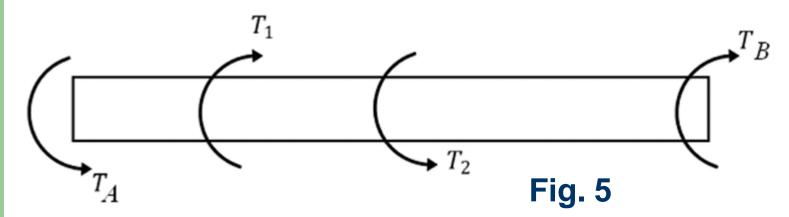


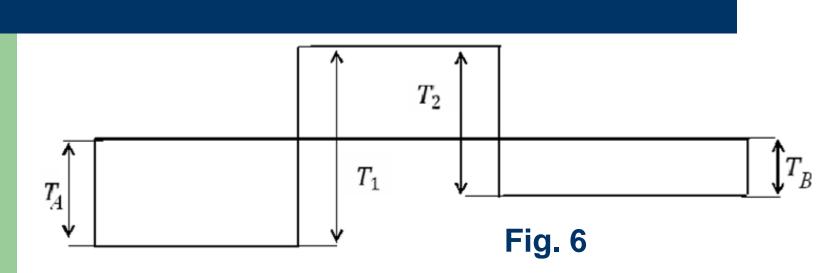
Fig. 4

Let us assume that the reactive torques and are positive in the directions shown in Fig. 5.



From static we have:

$$T_A - T_1 + T_2 - T_B = 0$$



we immediately have:

$$\theta_1 = \frac{T_A \ell_1}{GI_\rho} \quad \text{and} \quad \theta_2 = \frac{T_B \ell_3}{GI_\rho}$$

Hence, since the torque causing this deformation is $T_1 - T_A$, we have

$$\theta_1 + \theta_2 = \frac{(T_1 - T_A)\ell_2}{GI_{\rho}}$$

we find:

$$T_A = T_1 \frac{\ell_2 + \ell_3}{\ell}$$
 and $T_B = -T_1 \frac{\ell_1}{\ell} + T_2 \frac{\ell_1 + \ell_2}{\ell}$



Good bye!